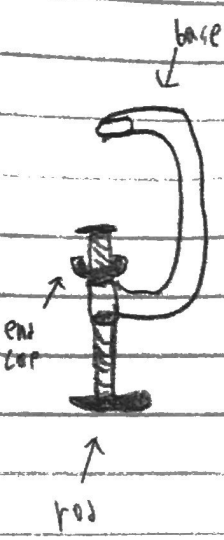


Sketches and part details

Ethan
Fredheim



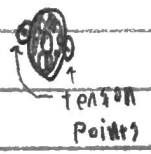
material \Rightarrow ^{304 stainless} steel $\mu_s = .5 - .8$ $\mu_k = .42$

Screw
 $3/8'' - 16$ $1\frac{1}{2}''$ Length Part#: 98980A977
 acme thread

(Rod) Tensile strength: 72,000 Psi

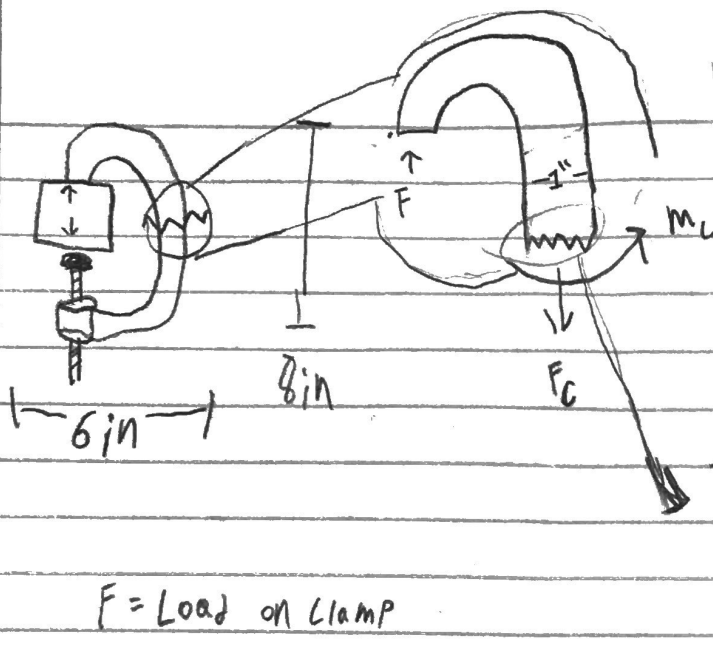
Travel distance / turn .03 inches

(cap) material
 steel



inside diam: $3/8''$
 outside diam: $7/8''$ Part #: 2207N12
 width: $3/8''$

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$\frac{3}{4}$ " depth

$$A = \frac{3}{4} \text{ in}^2$$

$F = \text{Load on clamp}$

Yield Tensile
Strength of
Steel = 50800 PSI



$$I_x = \frac{6(8^3) - 6^3(8-1)}{12} = 166 \text{ in}^4$$

Shear Modulus = 77×10^6 PSI

$$I_y = \frac{2(1)(6^3) + (6)(1)^3}{3} - \left((2)(6)(1) + 6(1) \right) \left(\frac{2(6^2)(1) + 6(1^2)}{2(6)(8) - 2(6)(5)} \right) = 106 \text{ in}^4$$

$$I = 272 = I_x + I_y$$

$$\text{Centroid} \approx \left(4, \frac{13}{6} \right) \quad \frac{13}{6} \left(\frac{3}{4} \right) = \frac{13}{8} = \left\{ \frac{13}{8} \right\} = Q \quad b = \frac{3}{4} \text{ in}^2$$

$$\tau = \frac{f Q}{I b}$$

Max Load would be dependent on clamp itself.
C-clamp body would fail before screw did